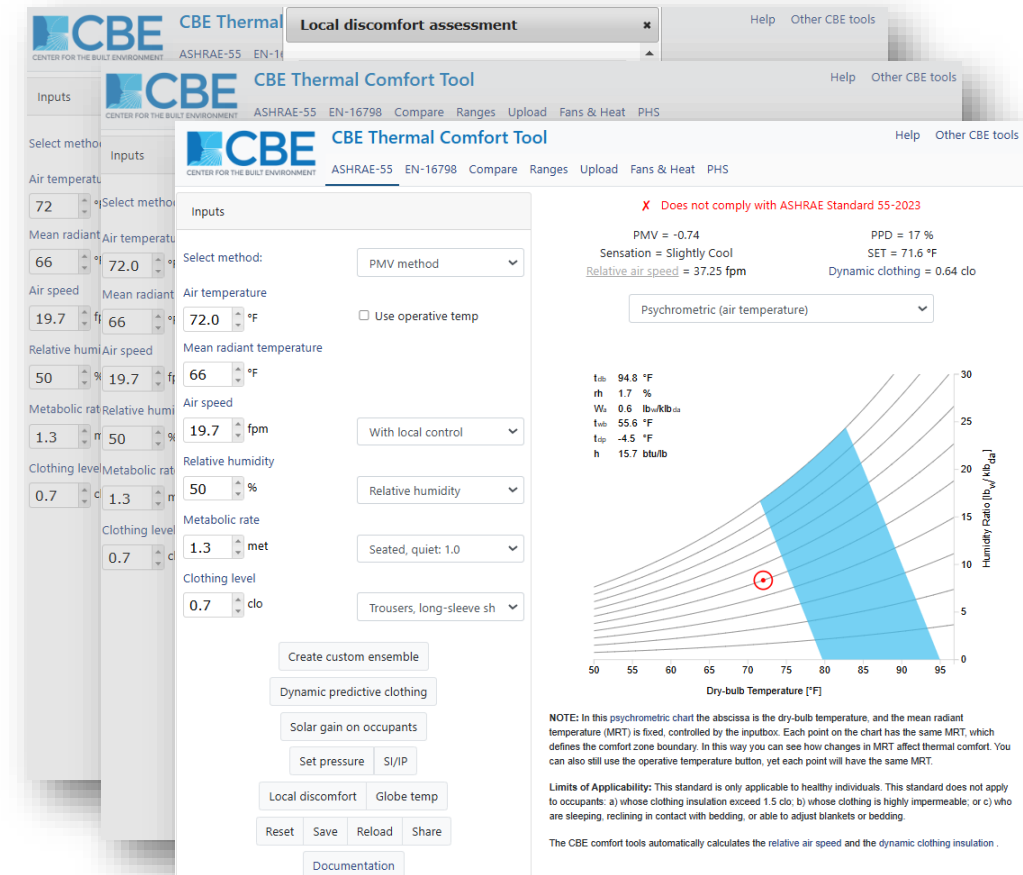


What's in the Thermal Comfort Tool?

Free Accessible Thermal Comfort Tool

- <https://comfort.cbe.berkeley.edu/>
- Inputs for
 - Personal Factors
 - General Factors
 - Local Factors
- Calculator
 - Clothing Predictor
 - Solar Gain Calculator
- Additional Calculators
 - Mean Radiant Temperature
 - Ceiling Fan Influencer
- Dual units (SI and I-P), examples shown in I-P
- [Click here to view Case Studies](#)



Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited



Thermal Comfort Tool : Case Study 1

Case Study 1

The entered air temperature (t_{db}) is similar to the MRT, indicating a higher-performing exterior zone or interior zone. The airspeed is unremarkable. The humidity is in the lower range. The met rate and clothing indicate a typical home office or office with a relaxed clothing policy.



Shows compliance

Occupant within the environmental boundaries of compliance.



Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

ashrae.org/IEQResources

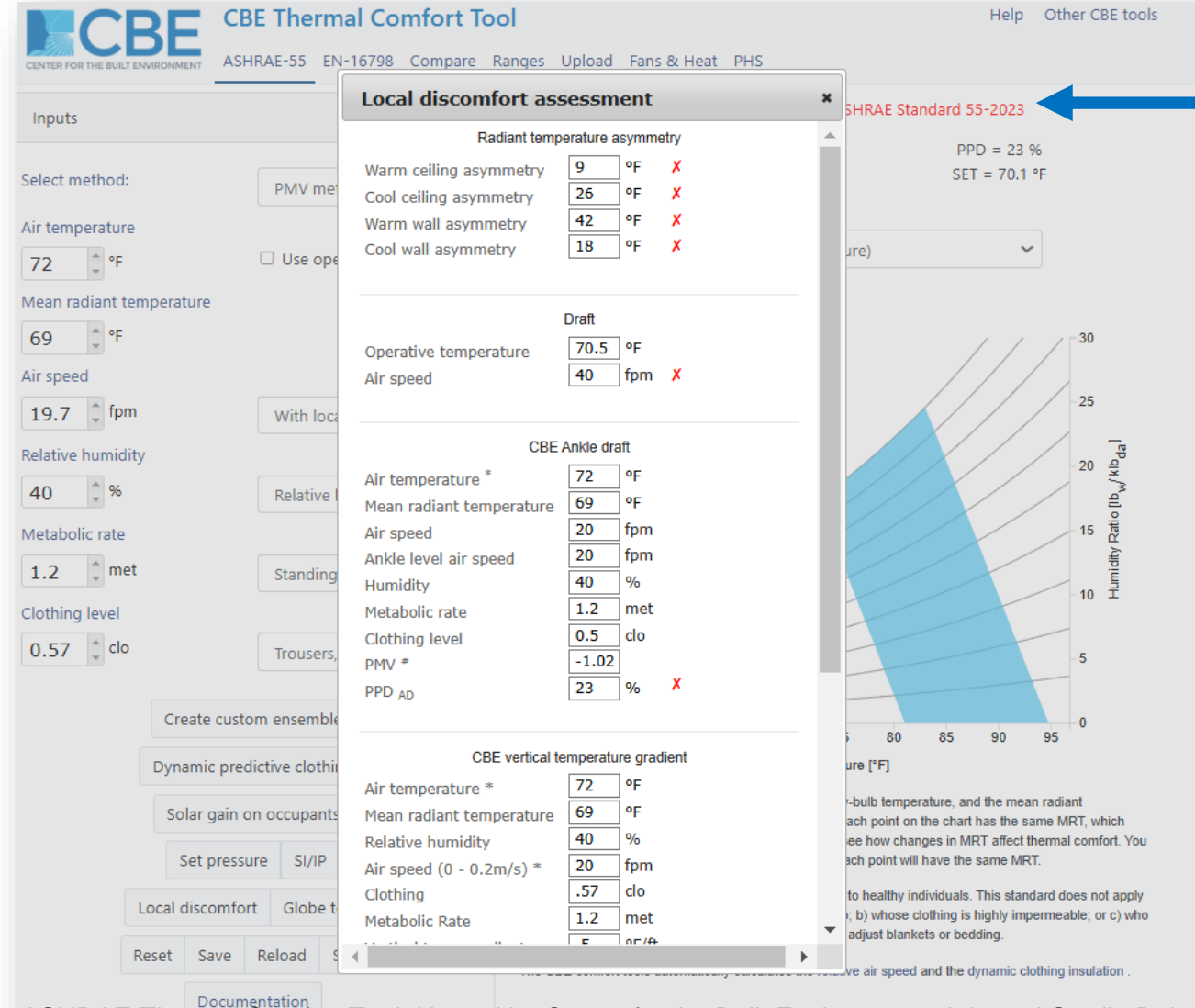
Thermal Comfort Tool : Case Study 1

Case Study 1 – Part 2a

Local discomfort assessment.

Threshold values for non-compliance are shown. Values can be calculated, measured, or simulated using CFD.

Ankle drafts are often associated with winter and are caused by warm air coming into contact with cold surfaces (glass), changing its buoyancy or displacement ventilation systems.



Shows non-compliance due to clothing and met rate



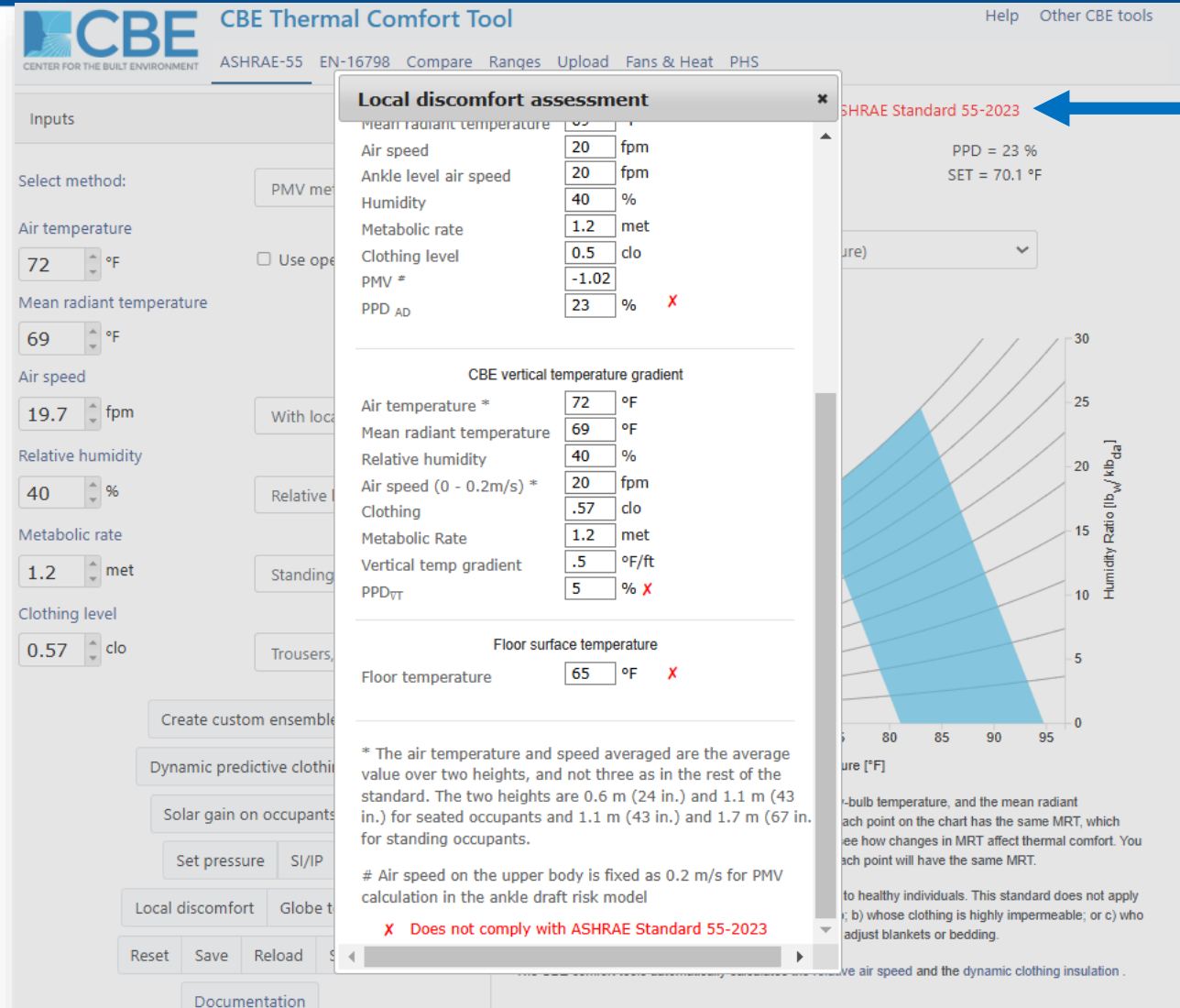
Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

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Thermal Comfort Tool : Case Study 1 *continued*

Case Study 1 – Part 2b

Local discomfort assessment. Vertical temp. differences are caused by buoyancy & stack effect & are associated with building performance & HVAC system type. Cool floor temps are influenced by back loss, usually due to inadequate or no insulation. High floor temps usually due to solar gains. Both can be managed with radiant floor systems.

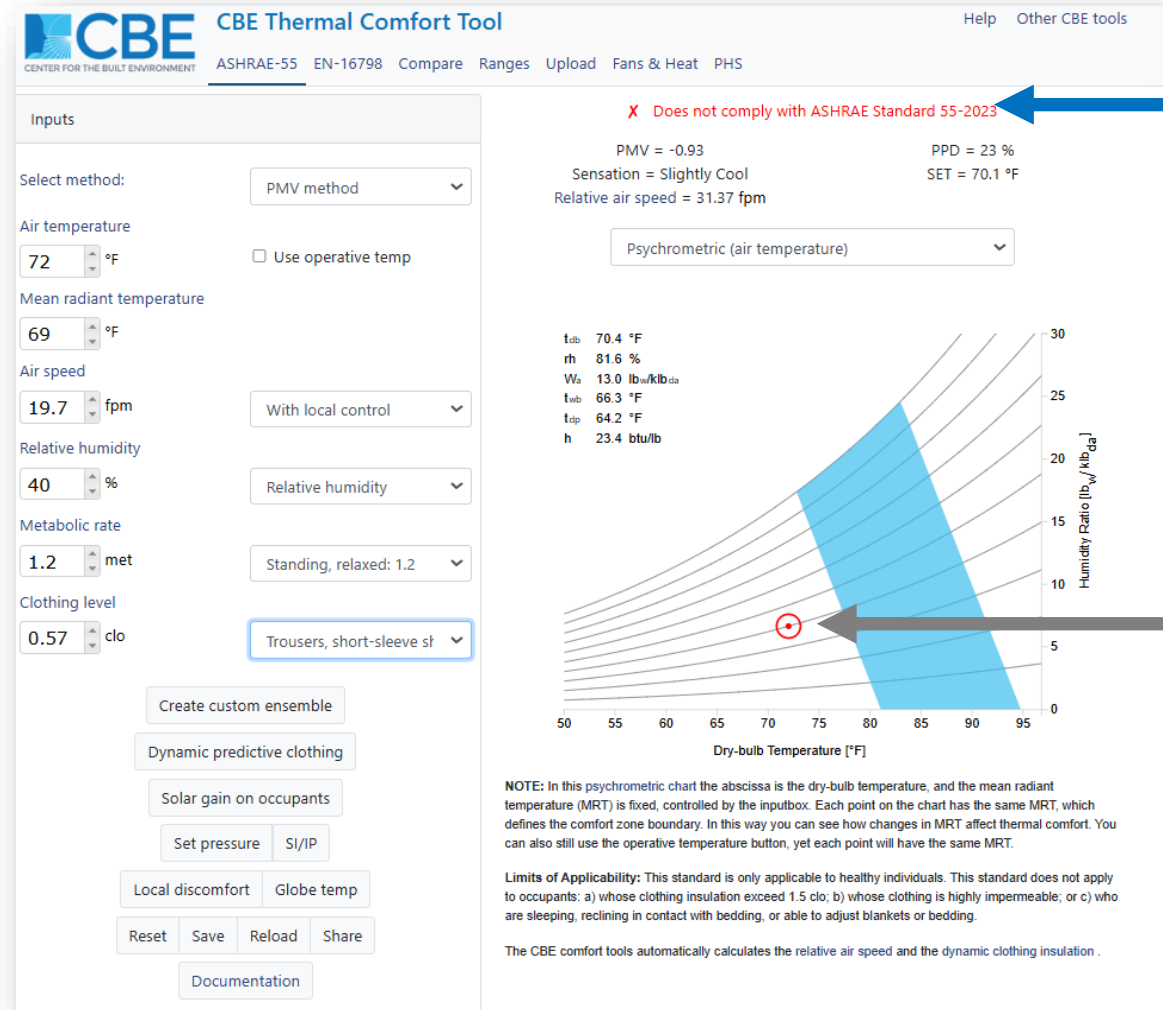


Shows non-compliance due to clothing and met rate

Thermal Comfort Tool: Case Study 2

Case Study 2 – Part 1

The MRT is lower than t_{db} , which is normal for buildings, even those with higher-performance enclosures, in cold to moderate climates. Airspeed is unremarkable. Humidity is in an acceptable range. Met rate is below 1.3, and clothing is below 0.7. In this scenario, local discomfort must be assessed.



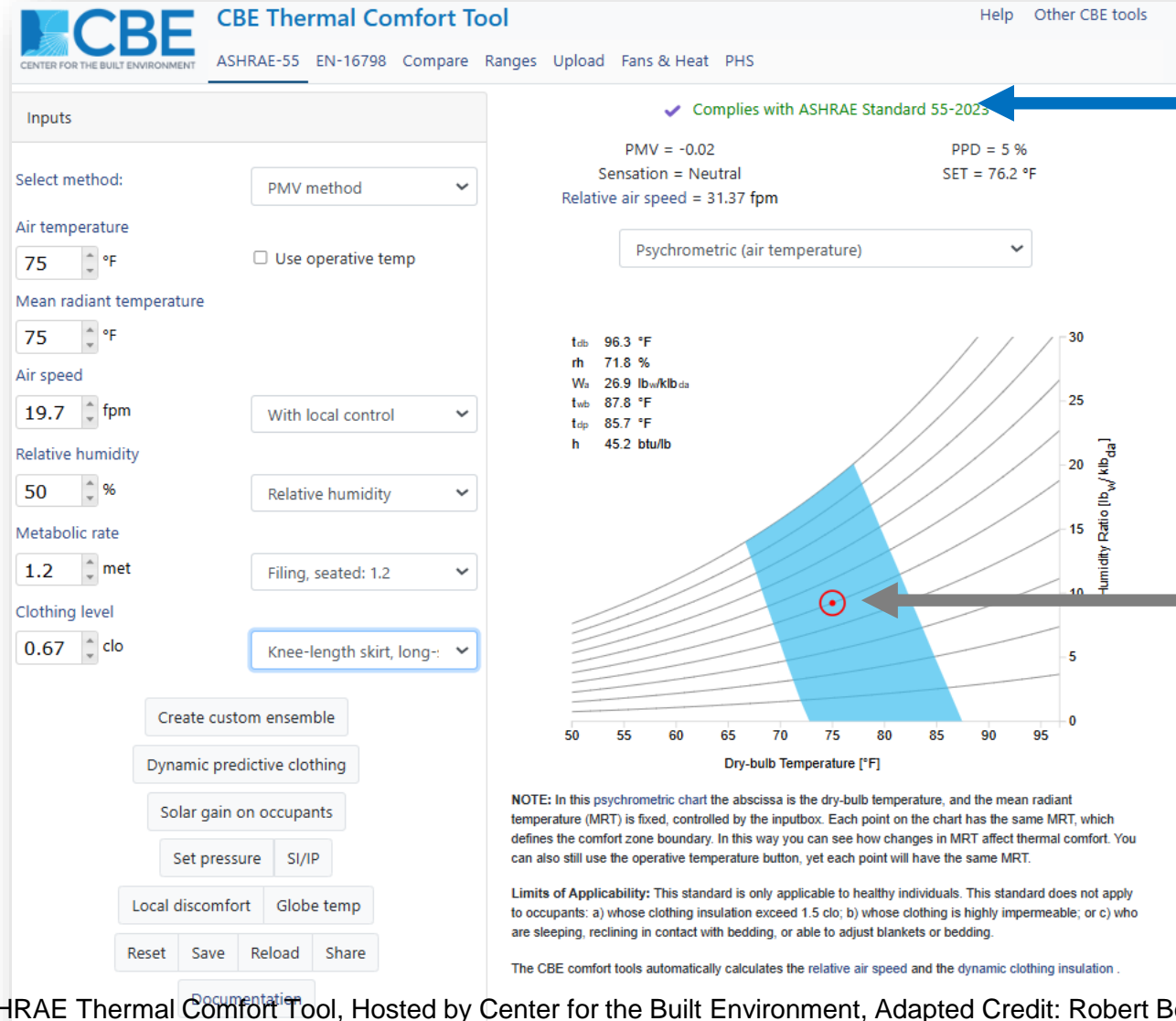
Shows non-compliance

Occupant outside the environmental boundaries of compliance.

Thermal Comfort Tool: Case Study 3

Case Study 3 – Part 1

The t_{db} value equals MRT, indicating a higher-performing exterior zone in a moderate climate or interior zone. The airspeed is unremarkable. The humidity is in the mid-preferred range. The met rate and clothing indicate a typical home office or traditional office...HOWEVER...what if there is influence due to solar gains through a window?



Shows compliance

Occupant within the environmental boundaries of compliance.



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Thermal Comfort Tool: Case Study 3 *continued*

Case Study 3 – Part 2

The solar calculator develops an adjusted mean radiant (MRT) due to shortwave radiation entering a space through a window and landing on an occupant. Here, the adjustment is 27.5°F (15°C)

The screenshot displays the CBE Thermal Comfort Tool interface. The main window shows input fields for Air temperature (75 °F), Mean radiant temperature (75 °F), Air speed (19.7 fpm), Relative humidity (50 %), Metabolic rate (1.2 met), and Clothing level (0.67 clo). The PMV method is selected, and the results show PMV = -0.02, Sensation = Neutral, PPD = 5 %, and SET = 76.2 °F. A blue arrow points to the text "Complies with ASHRAE Standard 55-2023".

A "SolarCal: shortwave radiation calculator" dialog box is open, showing the following inputs: Posture (Seated), Air temperature (75 °F), Mean radiant temperature (75 °F), Solar altitude above horizon (45 °), Solar horizontal angle relative to front of person (SHARP) (0 °), Direct beam (normal) solar radiation [I_{dir}] (800 W/m²), Total solar transmittance [T_{sol}] (.5), Sky vault view fraction [f_{svv}] (0.2), Fraction of body exposed to sun [f_{bes}] (0.6), and Average shortwave absorptivity [α] (0.85). The results show ERF : 63.8 W/m² and Mean radiant temperature delta : 27.5 °F. A grey arrow points to the "Mean radiant temperature delta" result.

The background shows a psychrometric chart with a blue shaded region representing the comfort zone. The chart axes are labeled "Humidity Ratio [lb_w/lb_{da}]" and "Temperature, and the mean radiant temperature, on the chart has the same MRT, which defines the comfort zone boundary. In this way you can see how changes in MRT affect thermal comfort. You can also still use the operative temperature button, yet each point will have the same MRT."

Below the chart, the "Limits of Applicability" are listed: "This standard is only applicable to healthy individuals. This standard does not apply to occupants: a) whose clothing insulation exceed 1.5 clo; b) whose clothing is highly impermeable; or c) who are sleeping, reclining in contact with bedding, or able to adjust blankets or bedding."

The CBE comfort tools automatically calculate the relative air speed and the dynamic clothing insulation.

Shows compliance without adjustment for shortwave radiation.

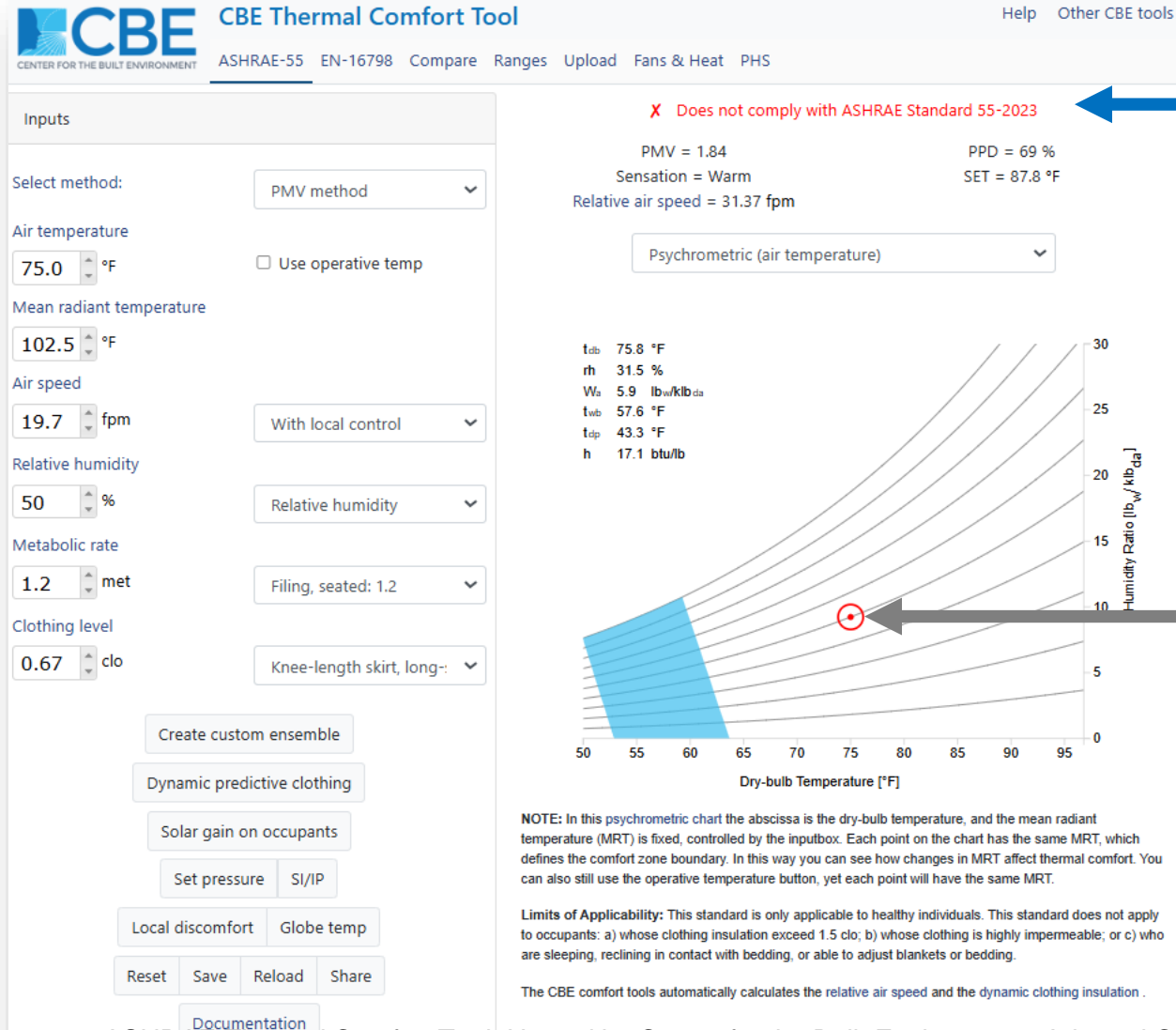
Mean radiant adjustment due to shortwave.



Thermal Comfort Tool: Case Study 3 *continued*

Case Study 3 – Part 3

When the MRT is adjusted by solar gains, the impact is non-trivial. It makes an otherwise compliant space noncompliant. This is why fenestration systems must be evaluated before design approval and construction. Consequently, buildings are operated cooler than necessary to compensate for solar gains.



Shows non-compliance when solar gain is considered.

Occupant outside the environmental boundaries of compliance.



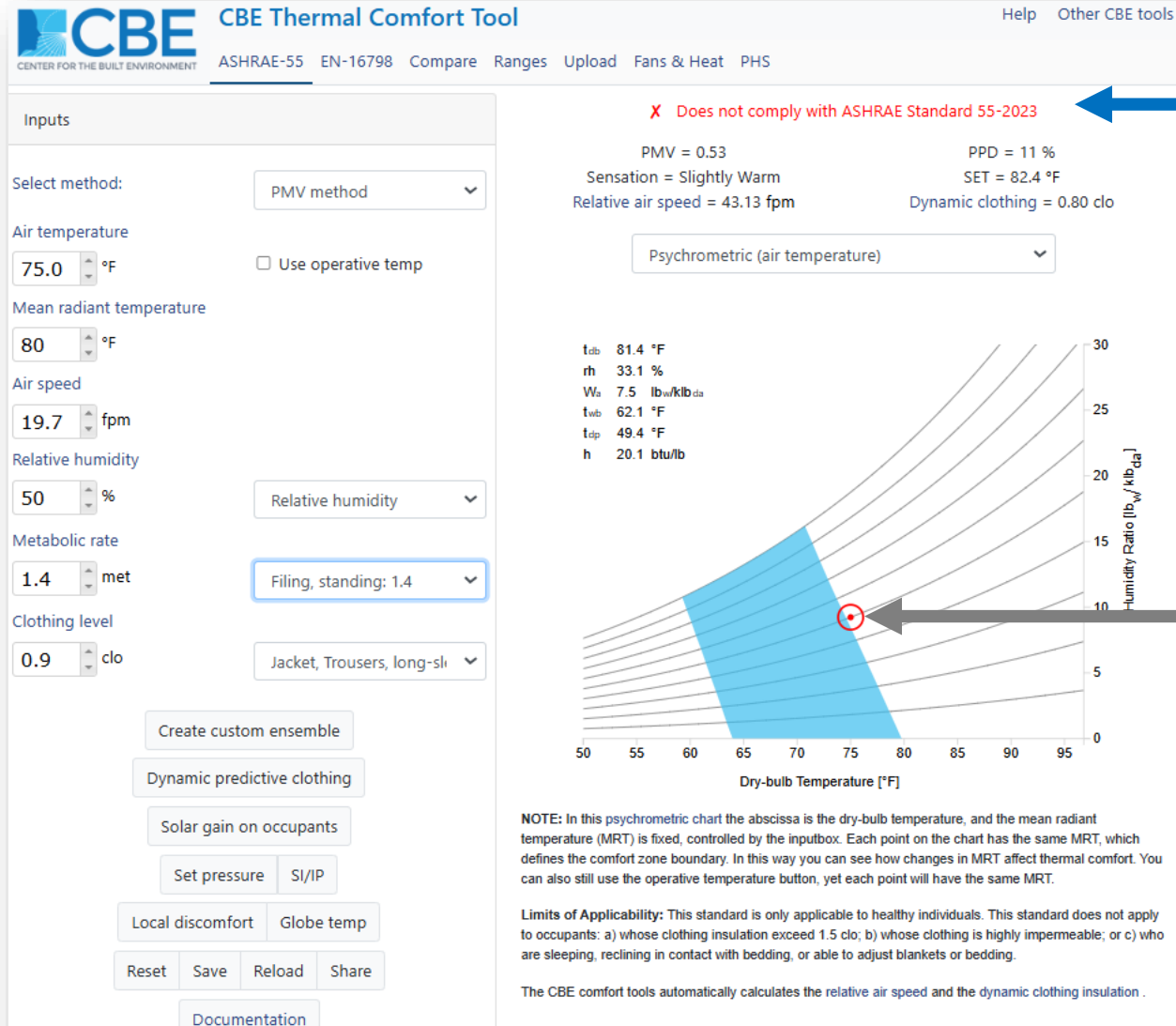
Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

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Thermal Comfort Tool: Case Study 4

Case Study 4 - Part 1

This is a summer scenario with higher t_{db} and MRT. Airspeed is unremarkable. Humidity is in the preferred mid range. However, higher met rates and office attire will lead to dissatisfaction and non-compliance due to the conditions. Option #1 is to dehumidify the space.



Shows non-compliance

Occupant outside the environmental boundaries of compliance.



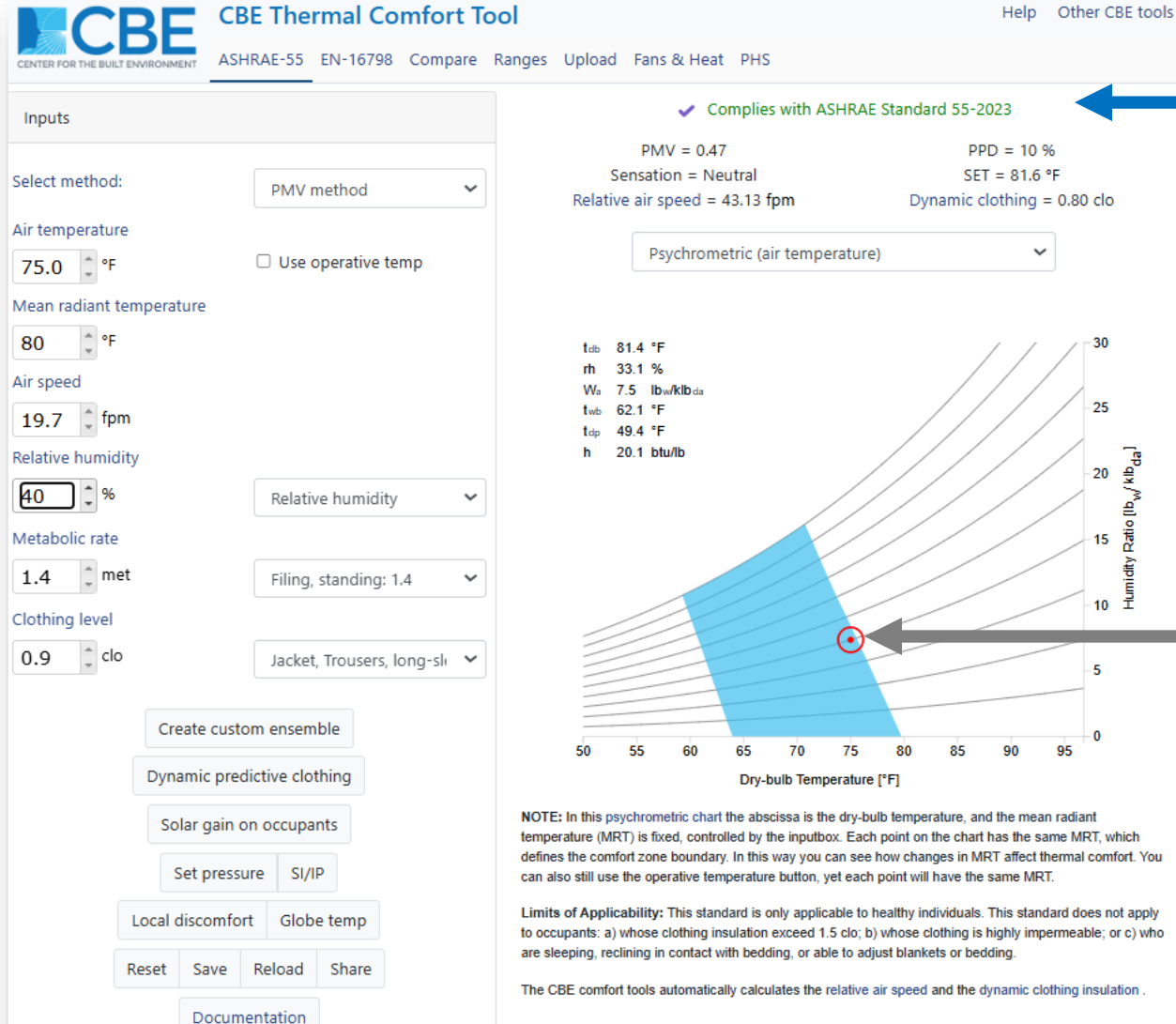
Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

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Thermal Comfort Tool: Case Study 4 *continued*

Case Study 4 – Part 2

By taking the humidity down by 10%, from 50% to 40%, there is a lower probability of occupant discomfort. Another option is to elevate the air speed across the occupants.



Shows compliance

Occupant inside the environmental boundaries of compliance.



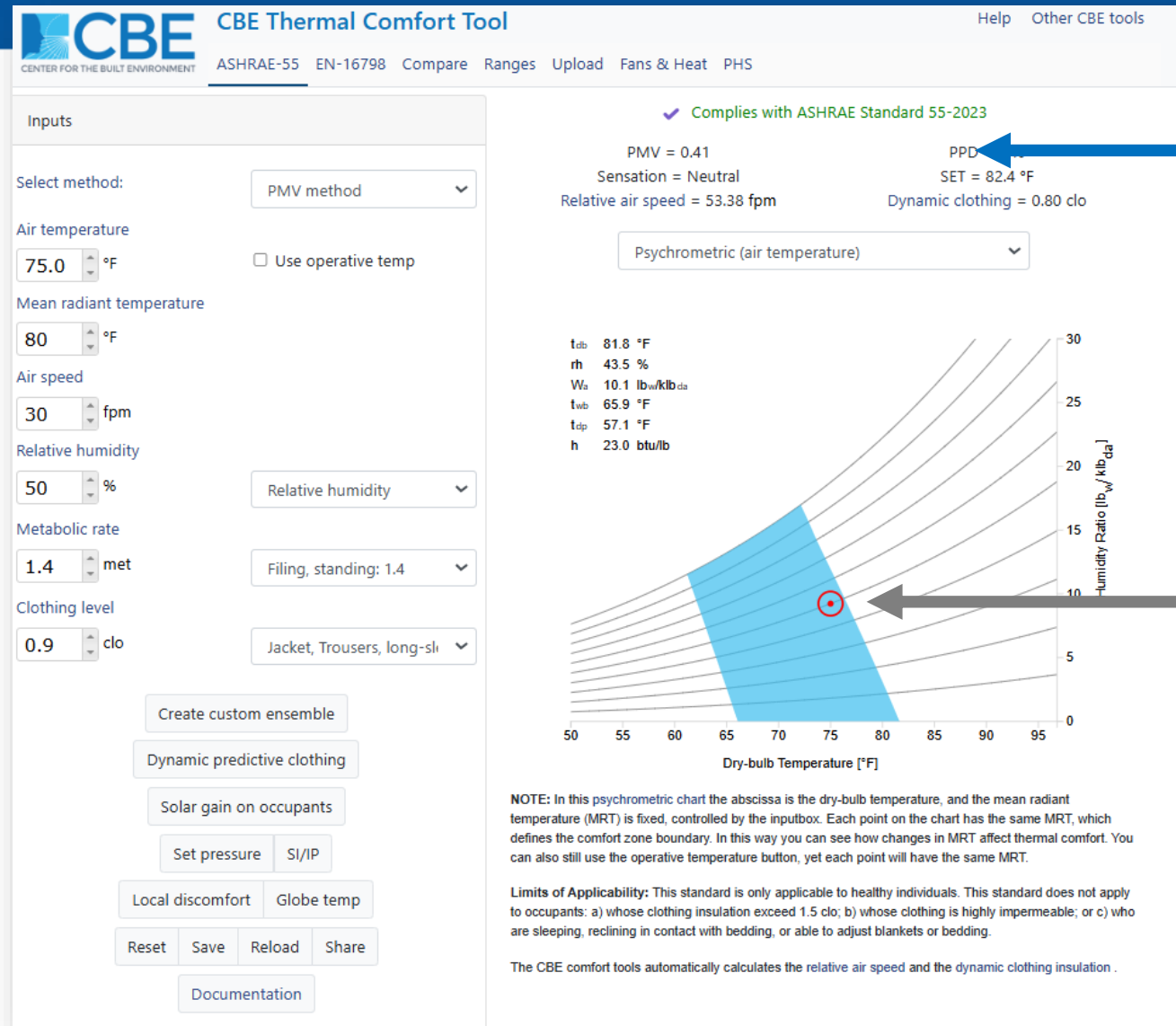
Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

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Thermal Comfort Tool: Case Study 4 *continued*

Case Study 4 – Part 3

By leaving the humidity at 50% and increasing the air velocity by 10 fpm, from 20 fpm to 30 fpm, there is a greater probability of compliance and occupant satisfaction. The designers and operators should then decide the pros/cons of adding or increasing dehumidification or adding ceiling fans (as an example)



Shows compliance

Occupant inside the environmental boundaries of compliance.



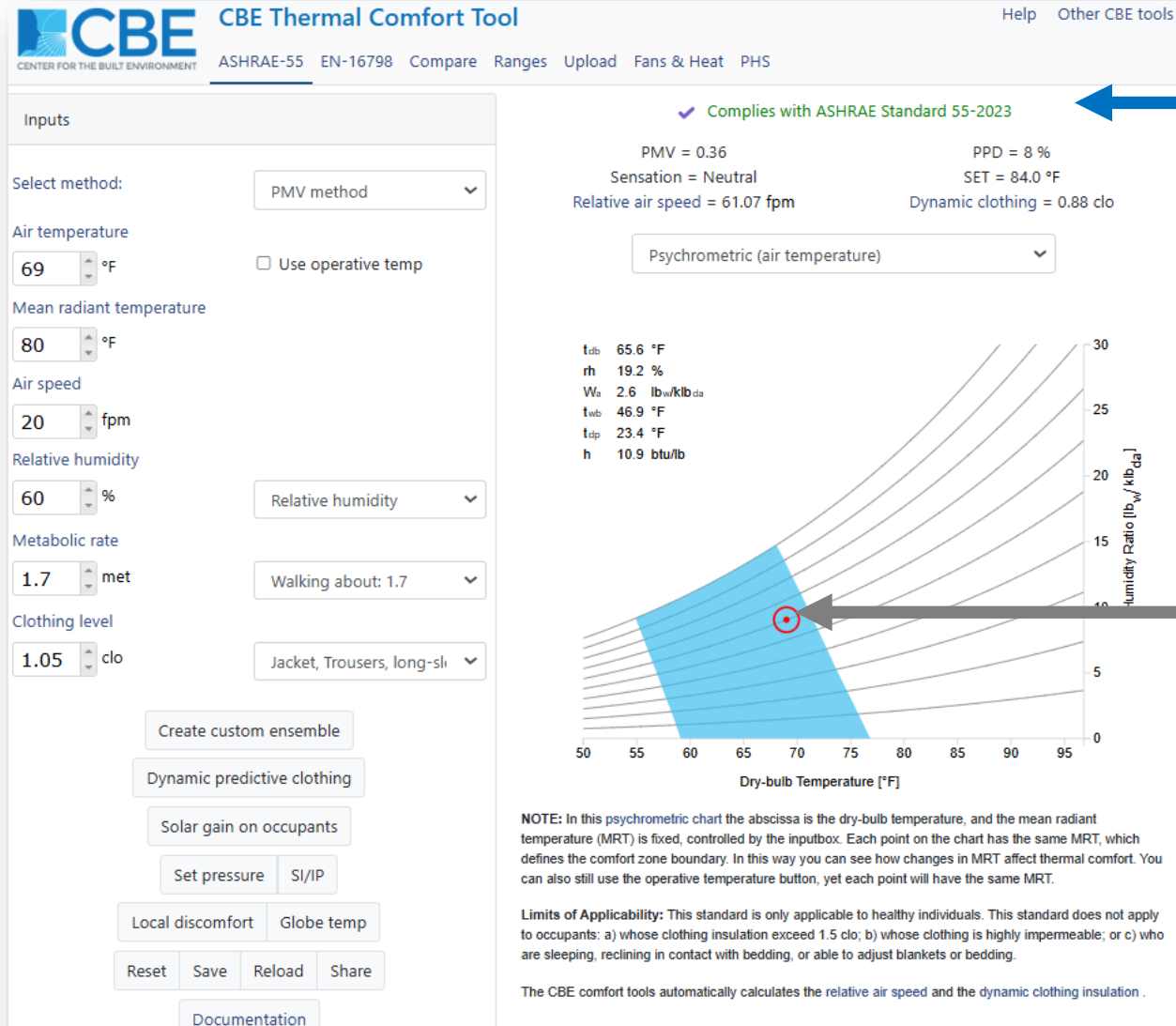
Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

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Thermal Comfort Tool: Case Study 5

Case Study 5 – Part 1

This is a summer example of an air-conditioned office with solar challenges. The office is maintained at an unnecessarily cooler temperature due to the workers' met rate (1.7) and clothing choices (1.05)....However, for those who wear lighter clothing, discomfort is highly likely.



Shows compliance

Occupant inside the environmental boundaries of compliance.



Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

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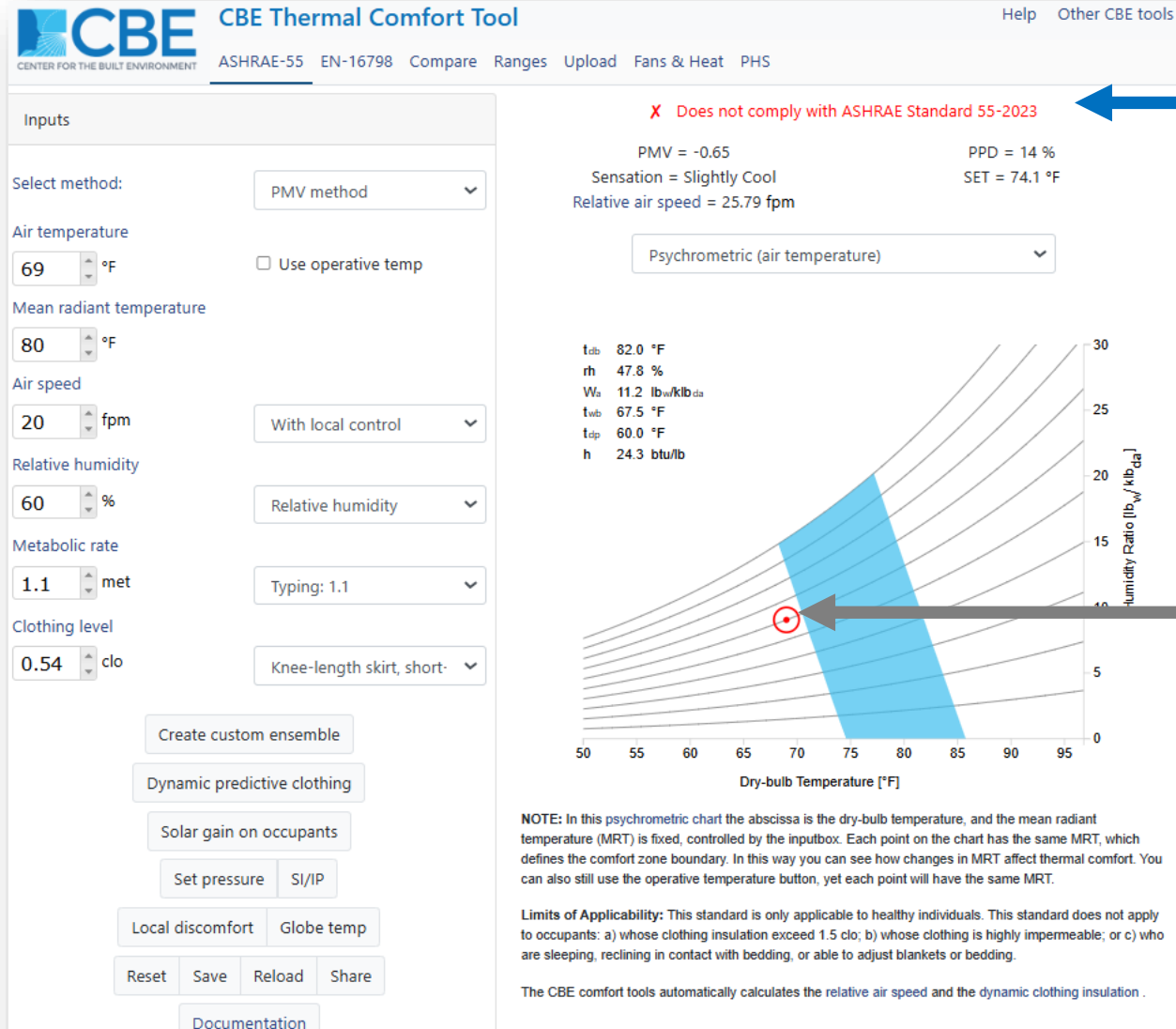
Thermal Comfort Tool: Case Study 5 *continued*

Case Study 5 – Part 5

As shown, those who wear lighter clothing (0.54) with lower met rates (1.1) will not be comfortable.

The option is to encourage lighter clothing for all occupants and let the space temperatures rise.

This not only improves comfort but also preserves and conserves energy.



Shows non-compliance

Occupant outside the environmental boundaries of compliance.



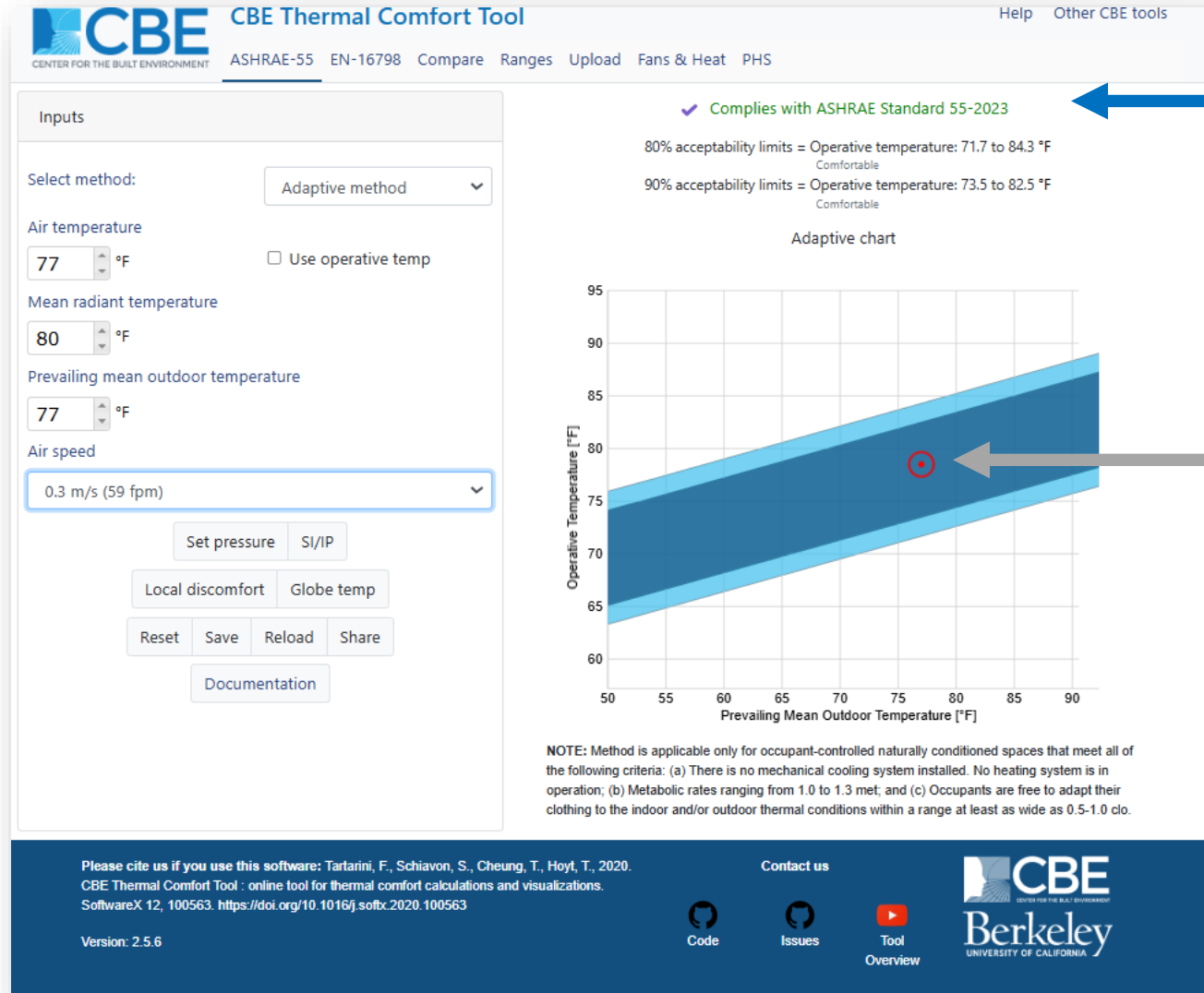
Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

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Thermal Comfort Tool: Case Study 6

Case Study 6

Adaptive comfort method is encouraged where and when it makes sense. Using this method, the outside is coupled to the inside. Practitioners should evaluate outdoor noise, air and light pollution and security risks when windows and doors are opened to the outdoors.



Shows compliance

occupant inside the environmental boundaries of compliance.

Graphic source: ASHRAE Thermal Comfort Tool, Hosted by Center for the Built Environment, Adapted Credit: Robert Bean, free use granted when credited

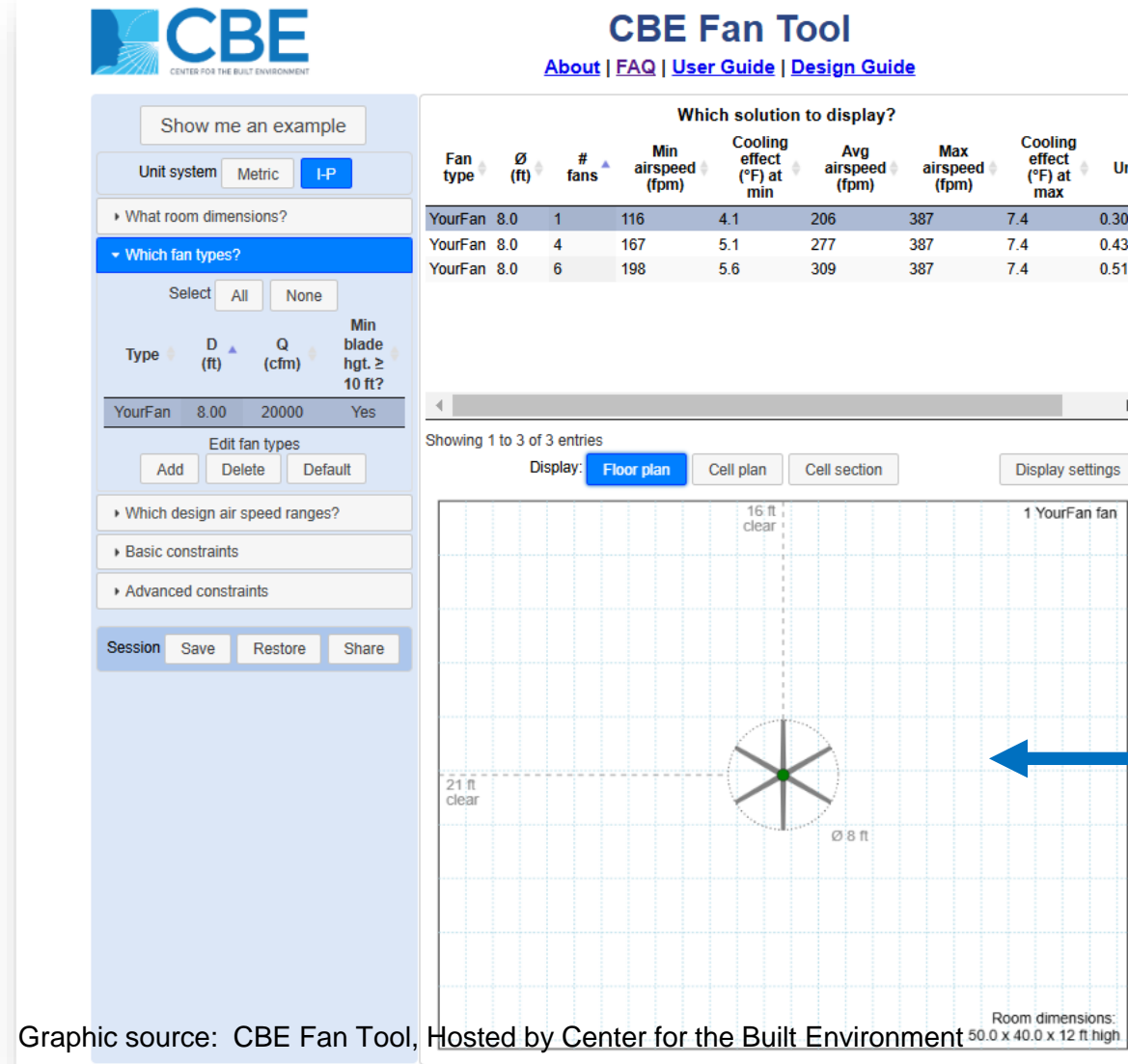
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Online Ceiling Fan Tool: Elevated Air Speeds

Using a ceiling fan simulator, the practitioner can create environments that are perceived to be cooler through increased convective flow across the body (cooling effect)...this effect can be ramped up, if necessary, by lowering the dry bulb temperature and if necessary, the humidity.

[Click here to view Case Studies](#)



Graphic source: CBE Fan Tool, Hosted by Center for the Built Environment

Make note of the cooling effect at min. and max. air flow.

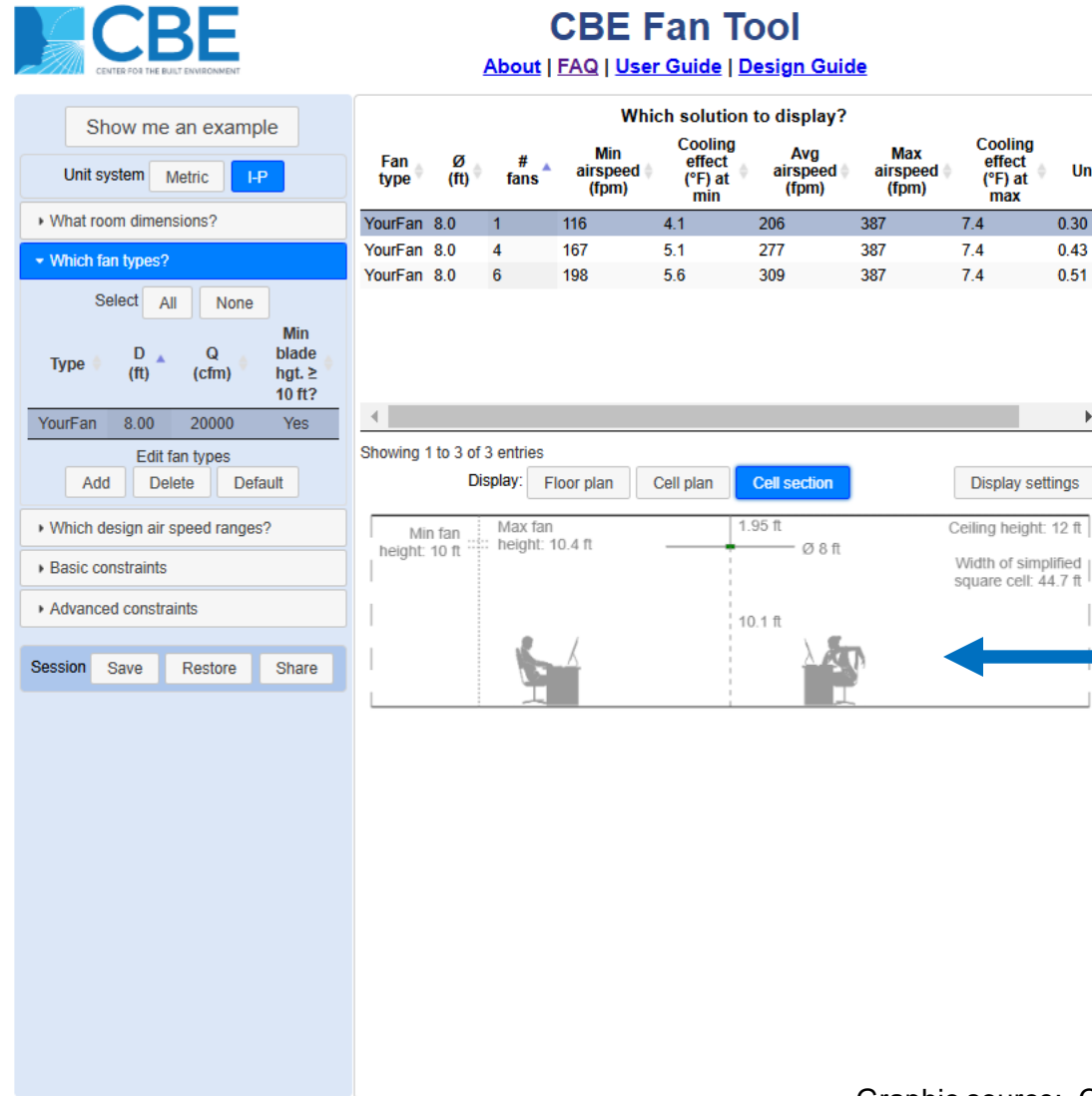
Fan-space geometry



Online Ceiling Fan Tool: Case Study 7 *continued*

Case Study 7

Using a ceiling fan simulator, the practitioner can create environments that are perceived to be cooler through increased convective flow across the body (cooling effect)...this effect can be ramped up, if necessary, by lowering the dry bulb temperature and if necessary, the humidity.



Single fan solution

Make note of the cooling effect at min. and max. air flow.

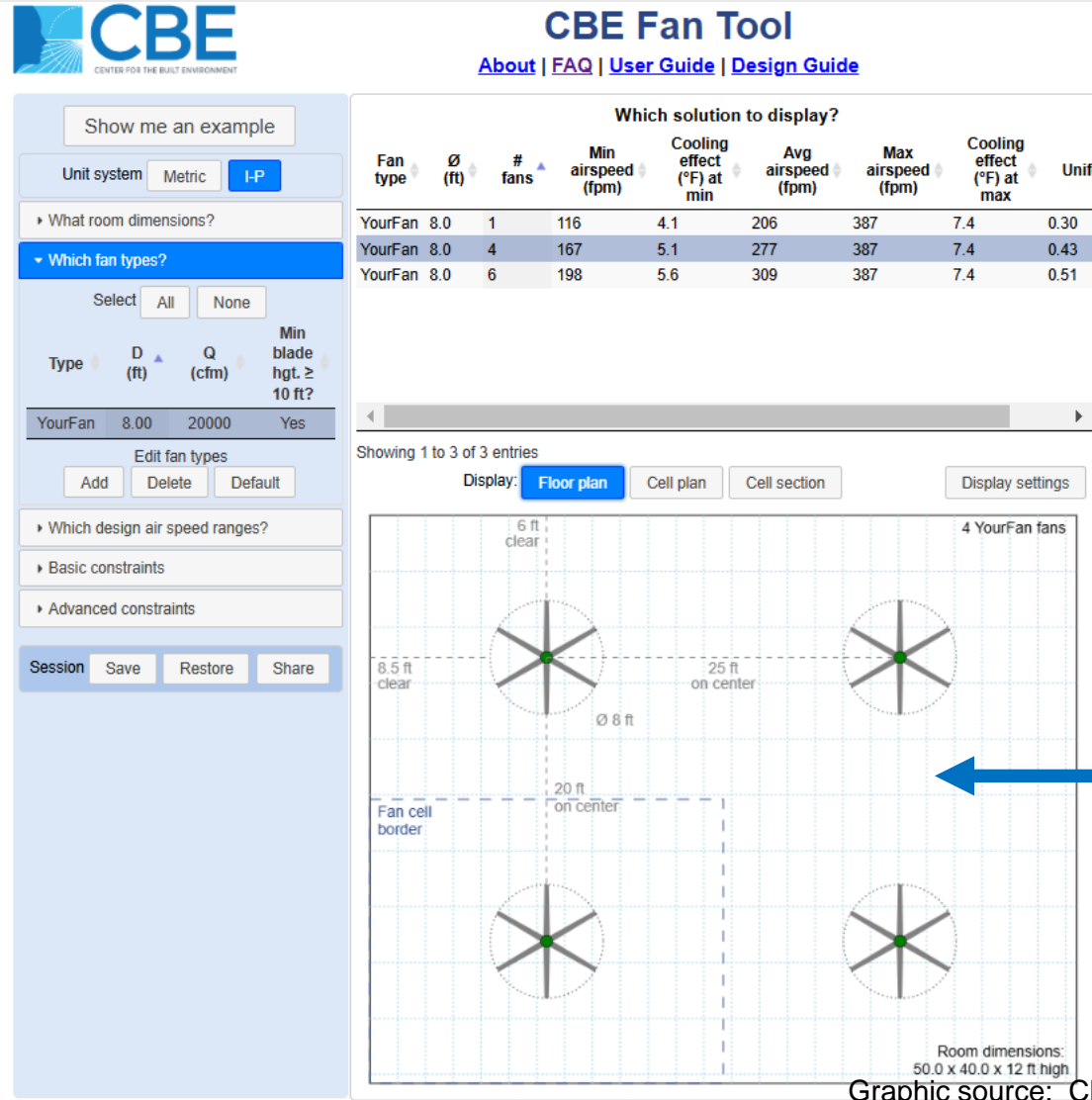
The sectional view of fan-space geometry.



Online Ceiling Fan Tool: Case Study 8

Case Study 8

Using a ceiling fan simulator, the practitioner can create environments that are perceived to be cooler through increased convective flow across the body (cooling effect)...this effect can be ramped up, if necessary, by lowering the dry bulb temperature and if necessary, the humidity.



Quad fan solution

Make note of the cooling effect at min. and max. air flow.

Fan-space geometry



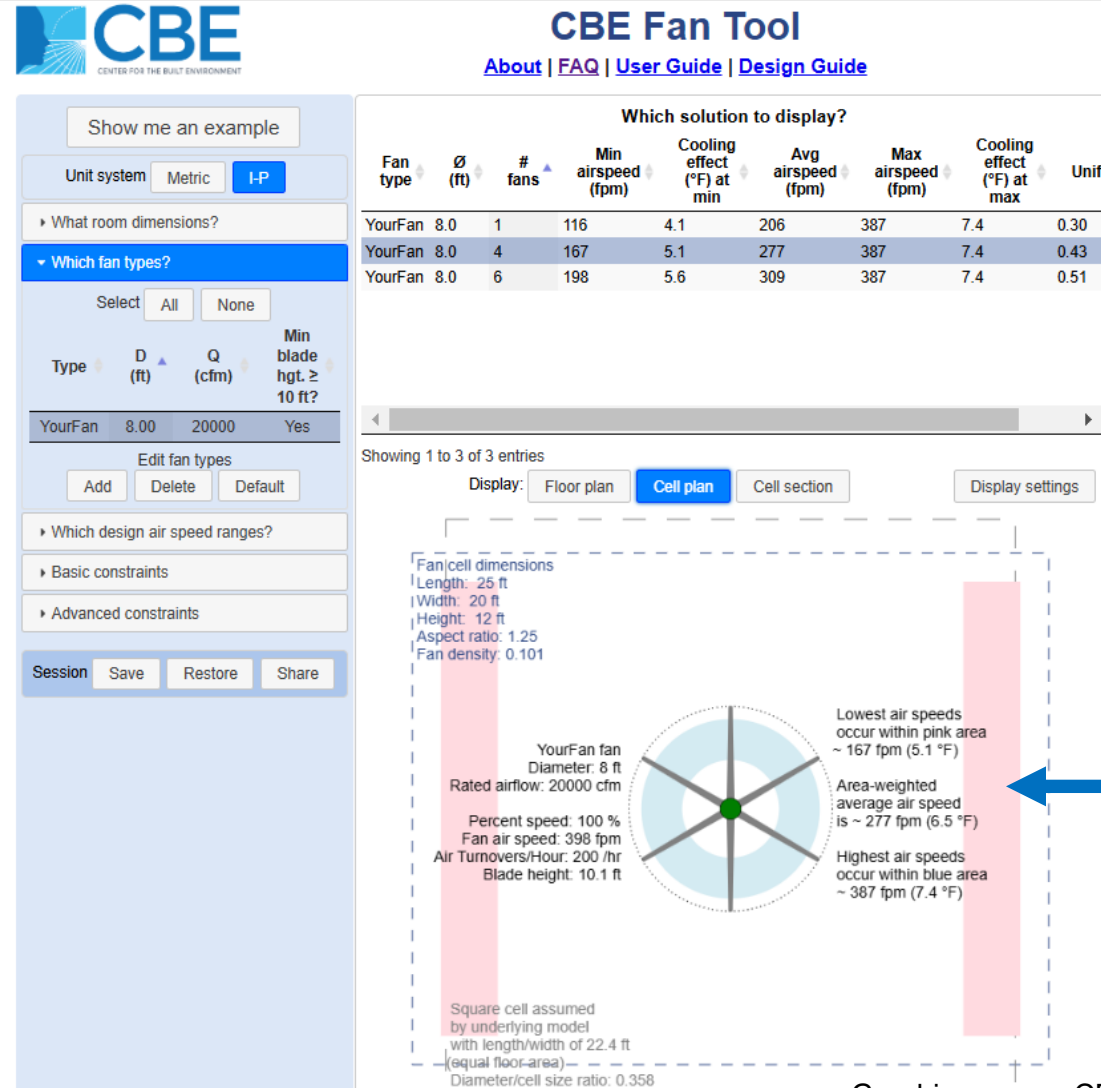
Graphic source: CBE Fan Tool, Hosted by Center for the Built Environment

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Online Ceiling Fan Tool: Case Study 8 *continued*

Case Study 8

Using a ceiling fan simulator, the practitioner can create environments that are perceived to be cooler through increased convective flow across the body (cooling effect)...this effect can be ramped up, if necessary, by lowering the dry bulb temperature and if necessary, the humidity.



Quad fan solution

Make note of the cooling effect at min. and max. air flow.

Make note of the cooling effect zones per quadrant.



Graphic source: CBE Fan Tool, Hosted by Center for the Built Environment

Online Ceiling Fan Tool: Case Study 8 *continued*

Case Study 8

Using a ceiling fan simulator, the practitioner can create environments that are perceived to be cooler through increased convective flow across the body (cooling effect)...this effect can be ramped up, if necessary, by lowering the dry bulb temperature and if necessary, the humidity.

CBE Fan Tool
[About](#) | [FAQ](#) | [User Guide](#) | [Design Guide](#)

Show me an example

Unit system: Metric **I-P**

What room dimensions?

Which fan types?

Select: All None

Type: D (ft) Q (cfm) Min blade hgt. ≥ 10 ft?

YourFan 8.00 20000 Yes

Edit fan types: Add Delete Default

Which design air speed ranges?

Basic constraints

Advanced constraints

Session: Save Restore Share

Which solution to display?

Fan type	Ø (ft)	# fans	Min airspeed (fpm)	Cooling effect (°F) at min	Avg airspeed (fpm)	Max airspeed (fpm)	Cooling effect (°F) at max	Unif
YourFan	8.0	1	116	4.1	206	387	7.4	0.30
YourFan	8.0	4	167	5.1	277	387	7.4	0.43
YourFan	8.0	6	198	5.6	309	387	7.4	0.51

Showing 1 to 3 of 3 entries

Display: Floor plan Cell plan **Cell section** Display settings

Min fan height: 10 ft Max fan height: 10.4 ft 1.95 ft Ø 8 ft Ceiling height: 12 ft Width of simplified square cell: 22.4 ft

10.1 ft

Quad fan solution

Make note of the cooling effect at min. and max. air flow.

The sectional view/quad of fan-space geometry.

