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ADDENDA

ANSI/ASHRAE Addendum o to ANSI/ASHRAE Standard 15-2024

Safety Standard for Refrigeration Systems

Approved by ASHRAE and the American National Standards Institute on September 30, 2025.

This addendum was approved by a Standing Standard Project Committee (SSPC) for which the Standards Committee has established a documented program for regular publication of addenda or revisions, including procedures for timely, documented, consensus action on requests for change to any part of the standard. Instructions for how to submit a change can be found on the ASHRAE® website (www.ashrae.org/continuous-maintenance).

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FOREWORD

Addendum o revises Informative Appendix C to fix an editorial error in an equation and revises the example calculation to use accurate thermodynamic properties of R-410A as calculated using REFPROP v10.0. It also revises Table C-1 to remove refrigerants whose use is banned under the Montreal Protocol and adds refrigerants approved by the U.S. Environmental Protection Agency under the SNAP program in Final Rules 21, 23, 25, and 26.

Informative Note: In this addendum, changes to the current standard are indicated in the text by <u>underlining</u> (for additions) and <u>strikethrough</u> (for deletions) unless the instructions specifically mention some other means of indicating the changes.

Addendum o to Standard 15-2024

Modify Section 9 as shown. The remainder of Section 9 is unchanged.

[...]

9.7.6* The rated discharge capacity of a *pressure relief device* expressed in lb of air/min (kg of air/s) shall be determined in accordance with ASME Boiler and Pressure Vessel Code¹⁵, Section XIII. When the relieving capacity of a *pressure relief device* is expressed in standard cubic feet per minute (scfm) of air, the density of air *shall* be set to 0.0764 lb/ft³. All pipe and fittings between the pressure relief valve and the parts of the refrigeration system it protects shall have at least the area of the pressure relief valve inlet area.

[...]

9.7.9.3.1 The design back pressure due to flow in the discharge *piping* at the outlet of *pressure relief devices* and *fusible plugs*, discharging to atmosphere, *shall* be limited by the allowable equivalent length of *piping* determined using Equation 9-7a or 9-7b:

 $[\ldots]$

where

[...]

 C_r = rated capacity as stamped on the pressure relief device in lb/min (kg/s), or in standard cubic feet per minute multiplied by 0.0764 lb/ft³ (the density of air in accordance with Section 9.7.6), or as calculated in Section 9.7.7 for a rupture member or fusible plug, or as adjusted for reduced capacity due to piping as specified by the manufacturer of the device, or as adjusted for reduced capacity due to piping as estimated by an approved method.

[...]

Modify Informative Appendix A as shown. The remainder of Informative Appendix A is unchanged.

[...]

Section 9.7.6

The specified value for the density of dry air of 0.0764 lb/ft³ is prescribed by the National Board of Boiler and Pressure Vessel Inspectors in their publication NB-18⁶⁸ to convert the calculated relieving capacity of pressure relief devices from pounds per minute to standard cubic feet per minute (scfm) of air. The National Board serves as the certification body for the pressure relief device requirements documented in the ASME Boiler and Pressure Vessel Code's Section XIII.

[...]

Modify Informative Appendix B as shown. The remainder of Informative Appendix B is unchanged.

 $[\dots]$

67. NIST. 2013. NIST REFPROP, Standard Reference Database 23, Version 9.1. National Institute of Standards and Technology, Gaithersburg, MD. Lemmon, E.W., I.H. Bell, M.L. Huber, and M.O. McLin-

den. 2018. NIST Standard Reference Database 23: Reference Fluid Thermodynamic and Transport Properties-REFPROP, Version 10.0. National Institute of Standards and Technology, Standard Reference Data Program, Gaithersburg, MD.

68. NBBI. 2024. *Pressure Relief Device Certification* (NB-18). Columbus, OH: National Board of Boiler and Pressure Vessel Inspectors (www.nationalboard.org).

Modify Informative Appendix C as shown. The remainder of Informative Appendix C is unchanged.

INFORMATIVE APPENDIX C METHOD FOR CALCULATING DISCHARGE CAPACITY OF POSITIVE DISPLACEMENT COMPRESSOR PRESSURE RELIEF DEVICE

[...]

$$\underline{W_v = \frac{Q \times PL \times \eta_v}{v_g}}$$
 (C-1)

$$W_r = \frac{Q \times PL \times \eta_v}{v_g} \tag{C-1}$$

where

 $W_r = \text{mass flow of } refrigerant, \text{ lb/min (kg/s)}$

[...]

Example

Determine the flow capacity of a pressure relief device for an R-410A compressor with a swept volume (Q) of $\frac{341-342}{9}$ ft³/min $(0.1609-0.1614 \text{ m}^3/\text{s})$. The compressor is equipped with capacity control that is actuated at 90% of the pressure relief device set pressure and has a minimum regulated flow of 10%.

 $Q = \frac{341}{342} \text{ ft}^3/\text{min}$

 $\eta_v = 0.90$, assumed volumetric efficiency of compressor

PL = 0.1, minimum regulated flow factor

$$v_g = (R-410A, T = 50^{\circ}F) = \frac{1.1979 \cdot 0.38214}{0.38214} \text{ ft}^3/\text{lb (I-P)}$$

 $v_g (R-410A, T = 10^{\circ}C) = 0.0748 - 0.023856 \text{ m}^3/\text{kg (SI)}$

$$\frac{W_r - \frac{341 \frac{\text{ft}^3}{\text{min}} \times 0.1 \times 0.9}{1.1979 \frac{\text{ft}^3}{\text{lb}}} - \frac{25.62 \frac{\text{lb}}{\text{min}}}{1.1979 \frac{\text{ft}^3}{\text{lb}}}$$
 (I-P [see C-1])

$$\frac{W_r = \frac{0.1609 \frac{\text{m}^3}{\text{s}} \times 0.1 \times 0.9}{0.0748 \frac{\text{m}^3}{\text{kg}}} = 0.1936 \frac{\text{kg}}{\text{s}}$$
(SI [see C-1])

$$W_a = W_r \times r_w = 25.62 \times 0.62 = 15.88 \frac{\text{lb}}{\text{min}} \text{ of air}$$
 (I-P [see C-2])

$$W_a - W_r \times r_w = 0.1936 \times 0.62 = 0.12 \frac{\text{kg}}{\text{s}} \text{ of air}$$
 SI [see C-2])

$$W_r = \frac{342 \frac{\text{ft}^3}{\text{min}} \times 0.1 \times 0.9}{0.38214 \frac{\text{ft}^3}{\text{lb}}} = 80.546 \frac{\text{lb}}{\text{min}}$$
 (I-P [see C-1])

$$W_r = \frac{0.1614 \frac{\text{m}^3}{\text{s}} \times 0.1 \times 0.9}{0.023856 \frac{\text{m}^3}{\text{kg}}} = 0.60890 \frac{\text{kg}}{\text{s}}$$
 (SI [see C-1])

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$$W_a = 80.546 \frac{\text{lb}}{\text{min}} \times 0.6208 = 50.003 \frac{\text{lb}}{\text{min}} \text{ of air}$$
 (I-P [see C-2])

$$W_a = 0.60890 \frac{\text{kg}}{\text{s}} \times 0.6208 = 0.37801 \frac{\text{kg}}{\text{s}} \text{ of air}$$
 SI [see C-2])

Converting to standard $\text{£}^3/\text{min}$ volumetric flow rate of air, $\dot{Q}_{a,std}$, where V_a —specific volume of air = 13.1 $\text{£}^3/\text{lb}$ (0.818 m^3/kg) for dry air at 60 °F (15.6 °C) $\rho_{da,std} = 0.0764 \text{ lb}_{m}/\text{ft}^3$ (1.224 kg/m³) for dry air at 60.0°F (15.6°C) at a barometric pressure of 14.70 psia (101.3 kPa):

 $\frac{\text{sefm}}{\text{sefm}} = \frac{13.1 \cdot (15.88) - 208.02 \text{ ft}^3 / \text{min (I-P)}}{\text{sefm}} = \frac{0.818 \cdot (0.12) - 0.098 \text{ m}^3 / \text{s (SI)}}{\text{min (I-P)}}$

$$\dot{Q}_{a, std} = W_a \times \frac{1}{\rho_{da, std}} = 50.003 \frac{\text{lb}}{\text{min}} \times \frac{1}{0.0764} \frac{\text{ft}^3}{\text{lb}_{\text{mda}}} = 654 \frac{\text{ft}^3}{\text{min}}$$
 (I-P)

$$\dot{Q}_{a, std} = W_a \times \frac{1}{\rho_{da, std}} = 0.37801 \frac{\text{kg}}{\text{s}} \times \frac{1}{1.224} \frac{\text{m}^3}{\text{kg}_{da}} = 0.309 \frac{\text{m}^3}{\text{s}}$$
 (SI)

Table C-1 Constants for Calculating Discharge Capacity

Refrigerant	k ^a	Relative Molar Mass	$C_{\mathbf{r}}$	$r_{ m w}$
R 11	1.137	137.4	330.7	0.49
R 12	1.205	120.9	337.7	0.51
R 13	2.050	104.5	403.4	0.46
R-22	1.319 <u>1</u>	86.5	348.8	0.5 <u>9850</u>
R 23	2.742	70.0	439.3	0.52
<u>R-32</u>	<u>1.5337</u>	<u>52.0</u>	<u>367.3</u>	<u>0.7164</u>
R 113	1.081	187.4	324.7	0.43
R 114	1.094	170.9	326.1	0.45
R-123	1.103 <u>5</u>	153.0 152.9	327.1	0.4 7 <u>691</u>
R-134a	1.196 <u>1</u>	102.0	336.8	0.5 6 <u>578</u>
<u>R-152a</u>	<u>1.2331</u>	<u>66.1</u>	<u>340.5</u>	<u>0.6854</u>
R-236fa	1.10 2 18	152.0	326.9	0.47 <u>08</u>
R-245fa	1.09 8 <u>78</u>	134.0	326.5	0.50 <u>21</u>
R-290	1.23 <u>6</u> 7	<u>44.044.1</u>	340.9	0.84 <u>383</u>
<u>R-600a</u>	<u>1.1346</u>	<u>58.1</u>	<u>330.5</u>	<u>0.7533</u>
<u>R-718</u>	1.3278	<u>18.0</u>	<u>349.6</u>	1.2795
<u>R-744</u>	<u>2.690</u>	<u>44.0</u>	<u>437.0</u>	<u>0.6546</u>
<u>R-1234yf</u>	<u>1.1685</u>	<u>114.0</u>	<u>334.0</u>	<u>0.5321</u>
<u>R-1234ze(E)</u>	1.1437	<u>114.0</u>	<u>331.4</u>	<u>0.5362</u>
R-404A	1.276 <u>5</u>	97.6	344.7	0.5 6 <u>572</u>
R-407C	1.269 <u>3</u>	86.2	344.1	0.59 <u>40</u>
R-410A	1.431 <u>0</u>	72.6	358.8	0.62 <u>08</u>
R 500	1.236	99.3	340.8	0.56
R 502	1.262	111.6	343.4	0.52
R-507A	1.282	98.9	345.2	0.55
R-600	1.122	58.1	329.2	0.76
R-718	1.328	18.0	349.6	1.28

a. Source: NIST REFPROP, Standard Reference Database 23, REFPROP v9.110.0, 20132018.67

Table C-1 Constants for Calculating Discharge Capacity

Refrigerant	k ^a	Relative Molar Mass	$C_{\mathbf{r}}$	$r_{ m w}$
R 744	2.690	44.0	437.0	0.65
<u>R-448A</u>	<u>1.2760</u>	<u>86.3</u>	<u>344.7</u>	0.5926
<u>R-449A</u>	<u>1.2755</u>	<u>87.2</u>	<u>344.7</u>	0.5896
<u>R-450A</u>	<u>1.1659</u>	<u>108.7</u>	<u>333.7</u>	0.5453
<u>R-452B</u>	<u>1.4324</u>	<u>63.5</u>	<u>358.9</u>	0.6635
<u>R-454A</u>	<u>1.2903</u>	<u>80.5</u>	<u>346.1</u>	<u>0.6112</u>
<u>R-454B</u>	<u>1.4299</u>	<u>62.6</u>	<u>358.7</u>	<u>0.6687</u>
<u>R-454C</u>	<u>1.2365</u>	<u>90.8</u>	<u>340.9</u>	<u>0.5842</u>
<u>R-455A</u>	<u>1.2435</u>	<u>87.5</u>	<u>341.6</u>	0.5939
<u>R-457A</u>	<u>1.2326</u>	<u>87.6</u>	<u>340.5</u>	0.6055
<u>R-513A</u>	<u>1.1859</u>	<u>108.4</u>	<u>335.8</u>	0.5428
<u>R-515B</u>	<u>1.1410</u>	<u>117.5</u>	<u>331.1</u>	0.5286
<u>R-516A</u>	<u>1.1849</u>	<u>102.6</u>	<u>335.7</u>	<u>0.5580</u>

a. Source: NIST REFPROP, Standard Reference Database 23, REFPROP v9.110.0, 20132018.67

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ASHRAE's short-range goal is to ensure that the systems and components within its scope do not impact the indoor and outdoor environment to a greater extent than specified by the Standards and Guidelines as established by itself and other responsible bodies.

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The effects of the design and selection of equipment and systems will be considered within the scope of the system's intended use and expected misuse. The disposal of hazardous materials, if any, will also be considered.

ASHRAE's primary concern for environmental impact will be at the site where equipment within ASHRAE's scope operates. However, energy source selection and the possible environmental impact due to the energy source and energy transportation will be considered where possible. Recommendations concerning energy source selection should be made by its members.

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